



Examine the Vapour Compression Refrigeration Cycle and Enhancement in Coefficient of Performance in VCRs

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Abstract — This paper focuses and affords literature studies on recent development in ejector or diffuser cooling machine additionally the enhancement of the overall performance. some of the researchers have conducted and classified in operating fluid choices, simulation and mathematical modeling of the ejector, geometrical and operation conditions optimization. but, maximum of the experimental research which has been performed in last many years are nonetheless insufficient if in comparison with simulation modeling; extra experiments studies and massive scale paintings are required in an effort to pop out with suitable information in real software.

Keywords- Vapor compression refrigeration cycle, Cop, Design, Performance, Diffuser, Ejector.

I. INTRODUCTION

All This paper focuses and provides literature studies on current development in vapor compression refrigeration. electrical power consumption has ended up a worldwide research subject matter because refrigeration and air-conditioning systems eating electric electricity about 15%. Ejector used vapor compression refrigeration device has some special benefits inclusive of the simplicity of creation, high reliability, and occasional cost evaluate to other refrigeration structures.

Vapor compression refrigeration device is based on vapor compression cycle. Vapor compression refrigeration machine is used in domestic refrigeration, food processing and bloodless storage, industrial refrigeration machine, shipping refrigeration and electronic cooling. So, development of the performance of gadget is too important for higher refrigerating effect or decreased energy consumption for identical refrigerating effect. Many efforts need to be achieved to enhance the overall performance of VC refrigeration gadget. Each product requires deferent garage temp for preserving the first-class of eatable or potable cloth. retaining that factor in view a low temp generating refrigeration device has been designed to preserve less then 0°C. it's far the technique of getting rid of warmth from an enclosed area, or from a substance, and rejecting it someplace else for the number one motive of decreasing the temperature of the enclosed space or substance after which keeping that lower temperature. The fast-technological development ends in a massive energy consumption. therefore, huge efforts of engineers are needed to limit electricity losses.

EU Union posted some regulations to force international locations to change standards in a machines layout to defend the herbal environment. Small vapor fridges are in commonplace use in the circle of relatives houses and farms. approximately 80% of general refrigeration strength is exploited through those small refrigerators. they may be, therefore, inside the organization of the machines of giant electricity consumption. Consequently, even a small development in those devices result in a large quantity of power saving. The reason for this research is to lower energy losses in such kind of devices.

In 1805, American inventor Oliver Evans described in the element, but never constructed, a refrigeration gadget based totally on the vapor compression refrigeration cycle. An American residing in extremely good Britain, Jacob Perkins, improved upon the design proposed with the aid of Oliver Evans and obtained the first patent. In 1842, an American health practitioner, John Gorrie, designed the first gadget for refrigerating water to supply ice. After reviewing the advancement in heat transfer era from an ancient attitude, the advantages of the use of a few special systems, the gadget can be made greater green and different cooling applications are mentioned, and research executed on diverse components of Vapor Compression Cycle performance is reviewed. This paper gives a roadmap of development in the thermal and fabrication components of VCRs.

II. REVIEW STAGE

As our challenge initiative is to have a look at primarily based so for element assignment study we referred many books and studies paper concerning our challenge for the designated study of the situation. The purpose of this examine is to design the components of a VCRs one after the other and to investigate the cycle. The effect of miniaturization on the pressure drop and the heat switch in condensation and evaporation strategies has been investigated. furthermore, an isentropic, reciprocating compressor has been designed for the gadget. Alternative evaporator designs with specific geometries were cautioned. The effect of poly-tropic compression process on the cycle COP has also been discovered.

To higher know-how of the way controllable variable effect on a compression refrigeration cycle performance, statically designed experiments have been carried out and information has been analyzed. consequences indicate that for inlet water temperature in the condenser, which ought to be set at low stage, the ultimate variable needs to be set at high

degree or closes it to maximize the cycle performance. From this paper, we located experimental setup and their calculation. And additionally, find overall performance parameters and what graphs necessary for our mission. [1]

Observe of Vapor Compression Refrigeration system the usage of diffuser and sub-cooling performed to enhance Coefficient Of performance (COP). And shows compression among new and present gadget. COP of new gadget will increase and rate of warmth rejection is decreased and size of the condenser will reduce. From this paper, I discovered that special manner of growing COP with diffuser and sub-cooling. [2]

VCR having an ejector, which could successfully save you warmth radiating ability of a radiator from being greatly reduced and this machine used for vehicle air-conditioner and layout of the ejector. with the aid of the usage of an ejector, in VCR cycle various mode of stress managed and compressor paintings relate and paper also suggests graphical conditioning of VCR at the extraordinary stage. Get a concept of machine layout, numerous graphical situations. And the criticism that I found from this paper is the design of ejector. [3]

In this Paper, by way of the test method with one of a kind pressure and temperature change find the most suitable situation. And it consists of that most of losses happens at beginning of compressor. And additionally, find the energy cycle depends on refrigerated mass. With the use of an adiabatic tank with right size and compressor, it helped to minimize losses. There are losses takes place in; compression due to wear; irreversibility in flow and increase entropy; electric powered power converted to mechanical strength so growth temperature of the cooled liquid; losses in the piston of the compressor. It beneficial for increase COP; a fee of common unit power strength decreased as a cooled liquid; For excessive cooled liquid require much less energy or given mass; discover the most fulfilling amount of cooled liquid for reducing energy loss. [4]

Warmness pump offers the gain of reduces electricity intake. right here examined warmth pump on single degree cycle, sub-cooling cycle and two-degree cycle, whilst tremendous displacement pump connects to warmth pump it will enhance cop; boom compressor efficiency; lessen the discharge; Ejector use for that reason KE of gasoline growth suction at compression and reduce irreversibility. [5]

Pressure replacing ejector is combined as ejector compressor with VCRs device. observe in mass fluid ratio, if the drop-in mass flow ratio greater fluid had to work with the circuit. additionally, this paper shows examine on experiment ratio and with a distinctive segment of liquid. In this research, the use of stress trade so as to minimize the entropy rise. that is used efficaciously where heat is to be had. And water is used as a refrigerant in this system efficaciously. there may be stress alternate among number one glide and secondary float via a nozzle. [6]

This paper suggests operating with VCRs to improve cop of the system. here theoretical analysis is offered and with

graphs and a few calculations, look at suggests that how can we enhance machine and make it extra higher and efficient. And results show that at very low evaporator temperatures, the COP turns into very low and additionally the length of the compressor turns into huge (because of small refrigeration impact). it can additionally be proven that the compressor discharge temperatures also increase as the evaporator temperature decreases. From this paper, we understand strain and temperature effect on cycle. [7]

This paper gives a roadmap for improvement in the thermal and fabrication components of micro-channels. From this, we're able to recognize warmness and Mass transfer in organic systems. [8]

This paper offers a detailed exergy analysis for theoretical vapor compression cycle the usage of R404A, R407C and R410A determination of most suitable conditions R-22 and R404a Refrigerant combinations in warmness Pumps the usage of Taguchi approach. The COP and efficiency of R407C are higher than that of R404A and R410A. The EDR of R410A is higher than that of R407C and R404A. This evaluation finished at condenser temperatures 40 °C and 50 °C and evaporator temperature ranging from 50 °C to 0 °C. [9]

III. MATHEMATICS AND FORMULAS

A. Refrigerating Effect :

= Amount of Heat absorbed in Evaporator.

$$Q_{evap} = (h_1 - h_4) + \text{Latent Heat} + \text{Superheated Heat} \quad (\text{kJ} / \text{kg})$$

B. Mass of Refrigerant :

= Amount of Heat absorbed / Refrigerating Effect.

$$m = \frac{14,000}{3600 (h_1 - h_4)} \quad (\text{kg} / \text{sec} - \text{tonne})$$

1 TR = 14,000 kJ/hr

C. Theoretical Piston Displacement :

= Mass of Refrigerant X Sp. Vol. of Refrigerant Gas $(v_g)_1$.

$$\text{Th. Piston Displ.} = \frac{14,000}{3600 (h_1 - h_4)} * (v_g)_1 \quad (\text{m}^3 / \text{sec} - \text{tonne})$$

A ton of refrigeration (TR), also called a refrigeration ton (RT), is a unit of power used in some countries (especially in North America) to describe the heat-extraction capacity of refrigeration and air conditioning equipment. ... A refrigeration ton is approximately equivalent to 12,000 BTU/h or 3.5 kW.

The standard unit of refrigeration is ton refrigeration or simply ton denoted by TR. It is equivalent to the rate of 32 0 F from water at 32 0 F in one day. The enthalpy of solidification of water from and at 32 0 F in British thermal unit is 144 Btu/lb. In general, 1 TR means 200 Btu of heat removal per minute. Thus, if a refrigeration system is capable of cooling at the rate of 400 Btu/min, it is a 2-ton machine. A machine of the 20-ton rating is capable of cooling at a rate of $20 \times 200 = 4000$ Btu/min. This unit of refrigeration is currently in use in the USA, the UK, and India. In many countries, the standard MKS unit of kcal/hr is used. If but ton unit is expressed into SI system, it is found to be 210 kJ/min or 3.5 kW. Refrigeration effect is an important term in refrigeration that defines the amount of cooling produced by a system. This cooling is obtained at the expense of some form of energy. Therefore, it is customary to define a term called coefficient of performance (COP) as the ratio of the refrigeration effect to energy input.

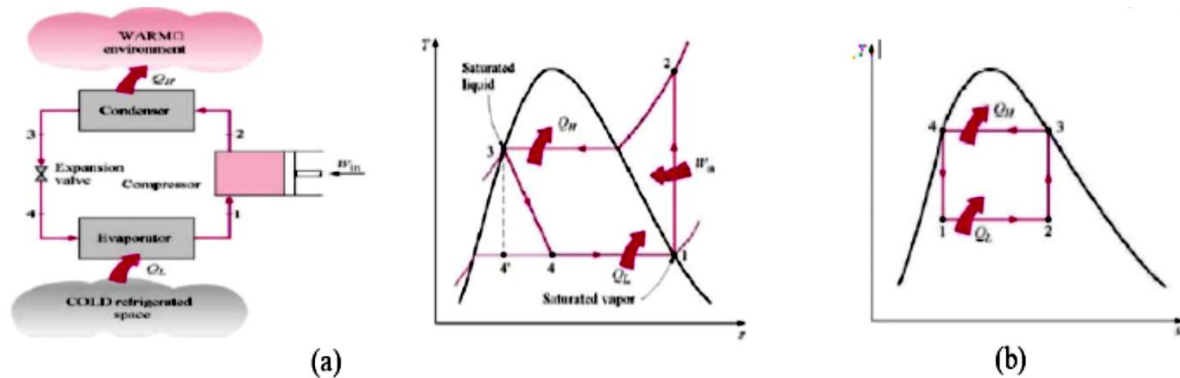


Figure 1: Vapour compression heat pumps with (a) Rankine and (b) Carnot cycles

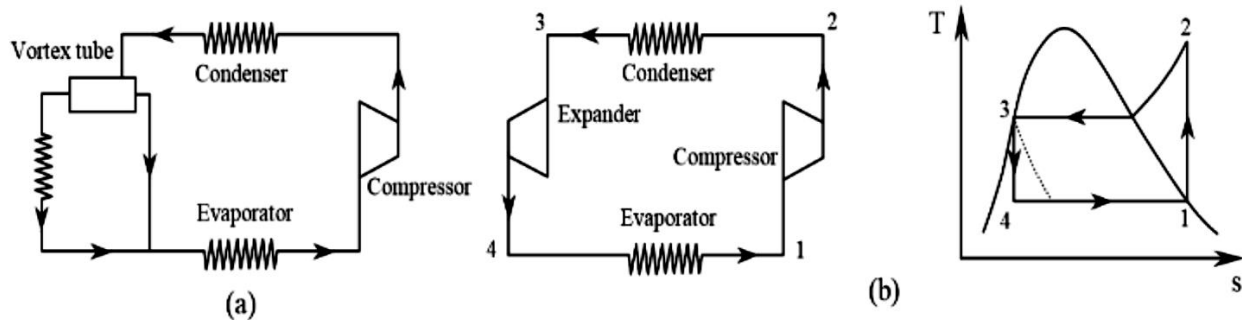


Figure 2: Cycles with (a) vortex tube and (b) with expander

IV. EQUATIONS

$$Q_{\text{evap}} = \dot{m}_{\text{water}} \cdot C_{p\text{water}} (T_{\text{outevap}} - T_{\text{evap}})$$

(2) Notation and variable units are as follows:

Q_{evap} —Refrigeration power (W); $\dot{m}_{\text{water}}$ —Water, mass flow rate in evaporator (kg/s);

$C_{p\text{water}}$ —Specific heat of water at constant pressure (4.18 kJ/kgK);

T_{evap} —Inlet water temperature in the evaporator (°C);

T_{outevap} —Outlet water temperature in the evaporator (°C).

The ratio between the Refrigeration (Q_{evap}) and Electric ($\dot{W}_{\text{elect}}$) powers is a metric used currently to assess the refrigeration cycles efficiency (Dabas, Dodeja, Kumar, & Kasana, 2011; Dincer, 2004; Mackensen, Klein, & Reindl, 2002; Pfister, 2004). This metric is called Coefficient of Performance (COP) and is defined as,

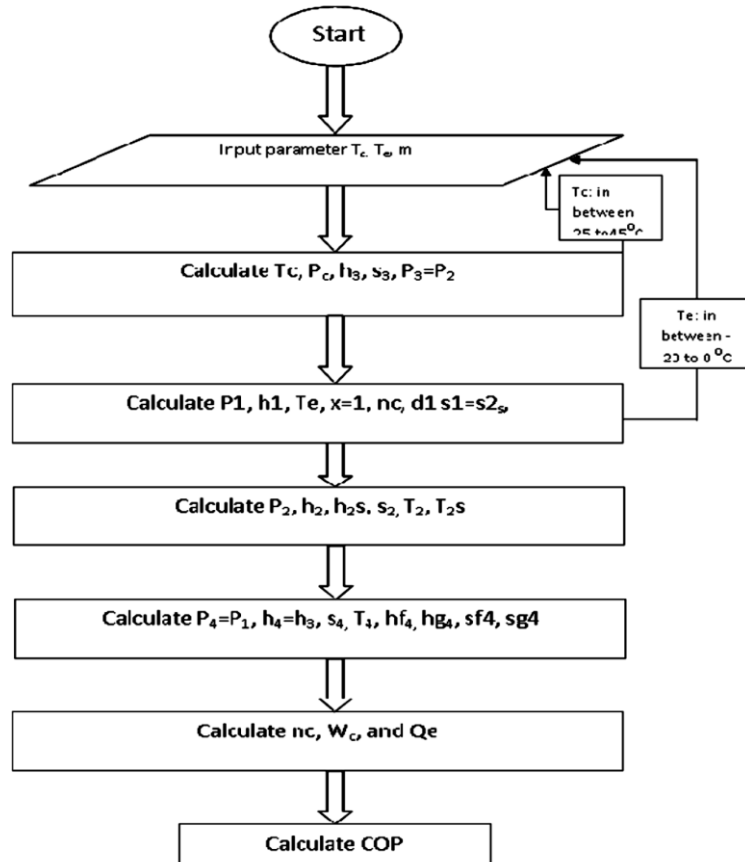
$$\text{COP} = Q_{\text{evap}} / W_{\text{elect}} \quad (1)$$

V. REFRIGERANT PROPERTY

No	Properties	R-134a
1	Boiling Point	-14.9°F or -26.1°C
2	Auto-Ignition Temperature	1418°F or 770°C

3	Ozone Depletion Level	0
4	Solubility In Water	0.11% by weight at 77°F or 25°C
5	Critical Temperature	252°F or 122°C
6	Cylinder Color Code	Light Blue
7	Global Warming Potential (GWP)	1200

VI. FLOWCHART FOR VAPOR COMPRESSION CYCLE PROCESS



VII. CONCLUSION

Various set of conclusions that are derived at the end of the work are as follows:

COP of VCR gadget has been as compared with and without a diffuser. it's been observed that COP with diffuser alongside with sub-cooling has been multiplied. the charge of heat rejection is also multiplied because of the increase in temperature.

The design and analysis of experiments to investigate refrigeration cycles are a singular approach to thermodynamics and effects give self-belief to use this method for refrigeration cycle layout and operation development purposes. therefore, as future research, we plan to use this system in the home and commercial system in addition to in different thermodynamic cycles. Simultaneous optimization of fashions suited for refrigeration and electric powered strength responses is another alternative optimization approach to maximize refrigeration cycle performance. to test different refrigeration fluids and compressor sorts can also be taken into consideration in future research studies. in this assessment, many opportunity vapor compression cycles carried out to warmth pumps were recognized and offered. these vary from simple to complicated cycles as well as from unmarried-stage to multi-level cycles, all emphasizing any stepped forward overall performance executed.

In preferred, the alternative cycles all provide several blessings to vapor compression warmth pumps which include reduced losses, extended performance, reduced strength consumption and many others. when as compared to the traditional simple vapor compression warmth pumps at the identical capability. Along with progressed overall performance, the complexity of the cycles and the fee of systems increase due to the multiplied quantity of ranges and the addition of new additives.

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